

Numerical investigation of liquid film flow on a rotating disk

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Outline

- 1 Motivation
- 2 Problem statement
- 3 Asymptotic solution
- 4 Numerical Simulation - VoF Method
- 5 Test cases
- 6 Results
- 7 Conclusions

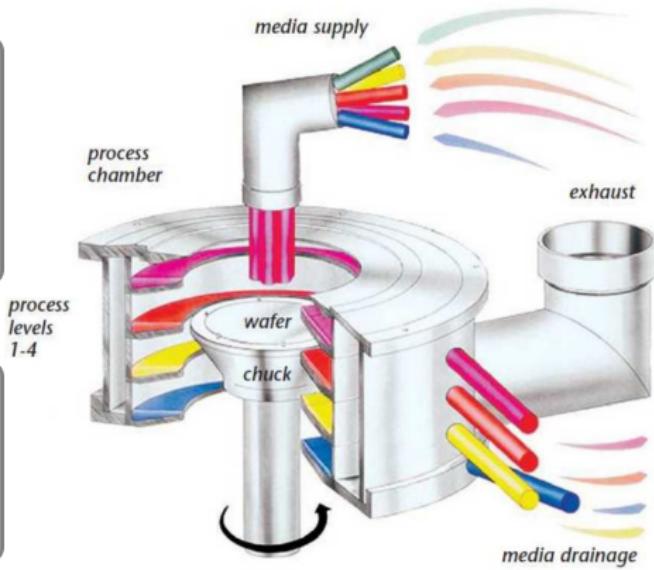
Motivation

Spin Processor Technology

- Single wafer one-sided etching
- Liquid supplied from above
- Rotating chuck

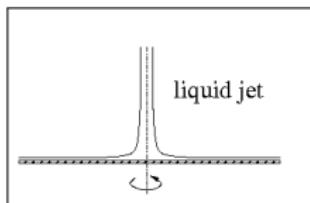
Objective

- Film flow *on* rotating surface
- CFD solvers:
FLUENT, OpenFOAM

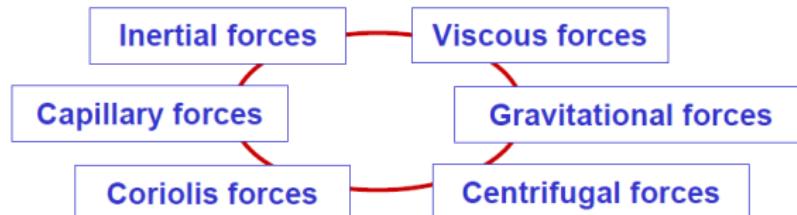


Problem statement

- Impinging jet on rotating disk



- Film motion governed by highly complex dynamics



Asymptotic solution

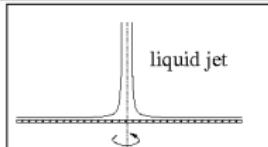
Nusselt solution

$$Ro^2 \ll 1, Ro^2 = \left(\frac{\bar{u}}{\omega r}\right)^2$$

$$\nu \frac{\partial^2 v_r}{\partial z^2} = -r\omega^2$$

Film thickness

$$\delta = \left(\frac{3}{2\pi} \frac{Q\nu}{\omega^2 r^2} \right)^{\frac{1}{3}}$$



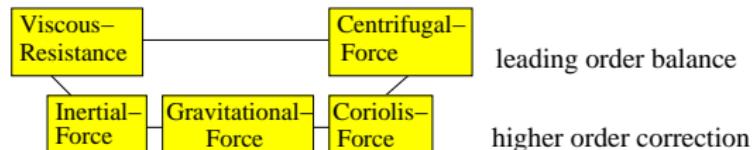
Asymptotic solution

Rauscher et al. (1973) [1]:

$$\frac{\delta}{h_0} = r^{*-2/3} + \left(\frac{62}{315} - \frac{2}{9} F^{-1} \right) r^{*-10/3} + \mathcal{O}(r^{-4})$$

$$\text{with } F^{-1} = \frac{2\pi g \nu}{3\omega^2 Q}, \quad r^* = r/l$$

characteristic lengths: $l = \left(\frac{9Q^2}{4\pi^2 \nu \omega} \right)^{\frac{1}{4}}$ and $h_0 = \left(\frac{\nu}{\omega} \right)^{\frac{1}{2}}$



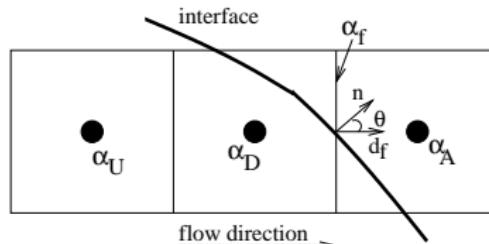
Numerical Simulation - VoF Method (Hirt, Nichols [3])

Volume fraction α

$$\alpha(\vec{x}, t) = \begin{cases} 1 & \text{liquid} \\ 0 & \text{gas} \\ 0 < \alpha < 1 & \text{2-phase zone} \end{cases}$$

Advection equation ($\nabla \cdot \vec{u} = 0$)

$$\frac{\partial \alpha}{\partial t} + \nabla \cdot (\alpha \vec{u}) = 0$$



Surface tracking

Interpolation of face values:

- boundedness criterion
- preserve sharp interface

Surface tracking methods

- **Higher Order Differencing**
(HRIC, Inter- γ , QUICK, ...)
- **Reconstruction Schemes**
(PLIC, ...)

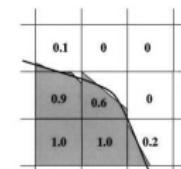


Figure adopted from [2]

Test cases

(Experiments: Thomas et al. 1991, Ozar et al. 2003)

Radially injected liquid sheet

Volumetric flowrate Q ,

rotational speed ω

and δ_0 prescribed.

Inner radius: $r_1 = 50.8\text{mm}$,

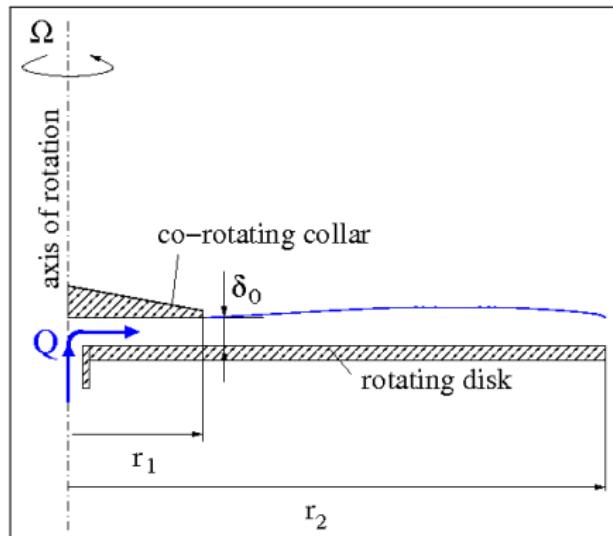
outer radius: $r_2 = 203\text{mm}$.

- Test case I:

$\omega = 200\text{rpm}, Q = 7\text{lpm}$

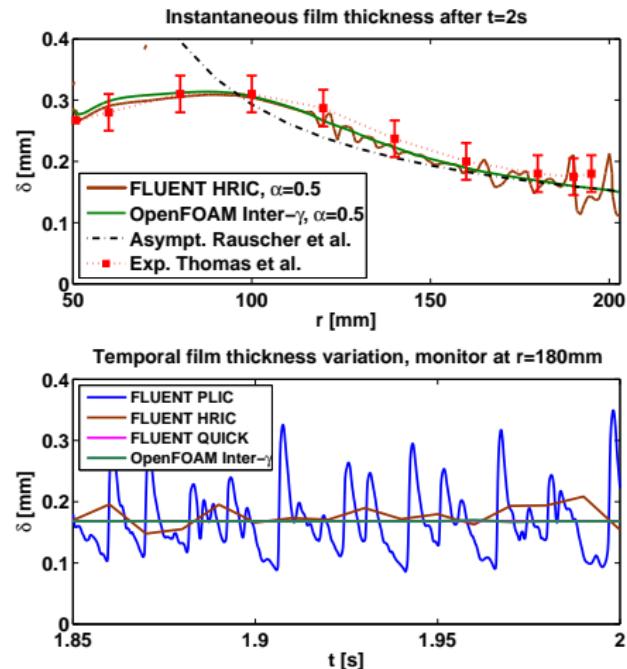
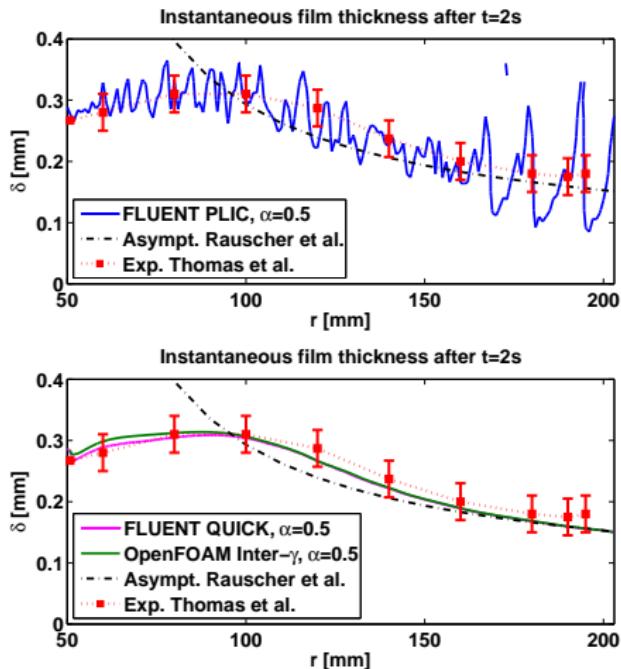
- Test case II:

$\omega = 300\text{rpm}, Q = 3\text{lpm}$

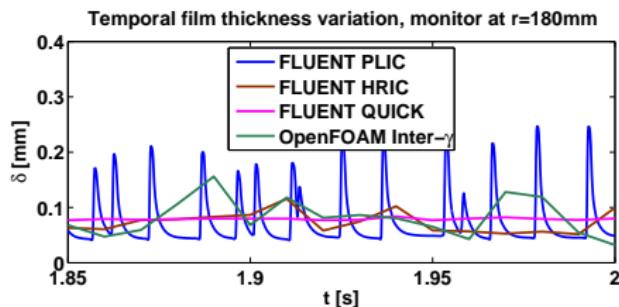
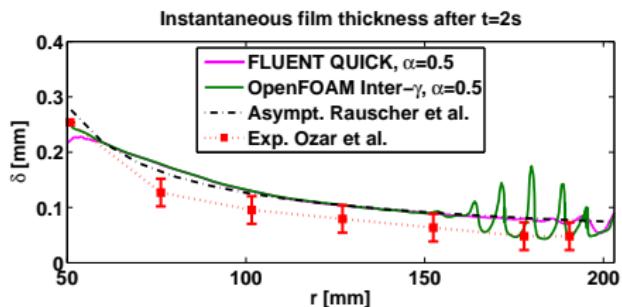
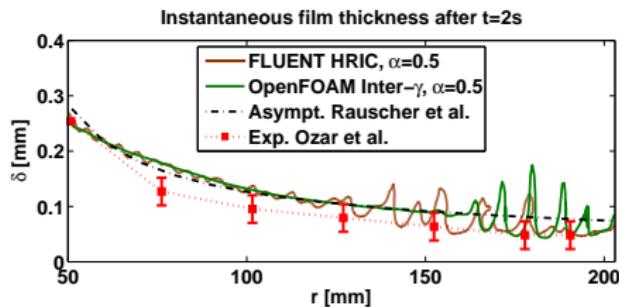
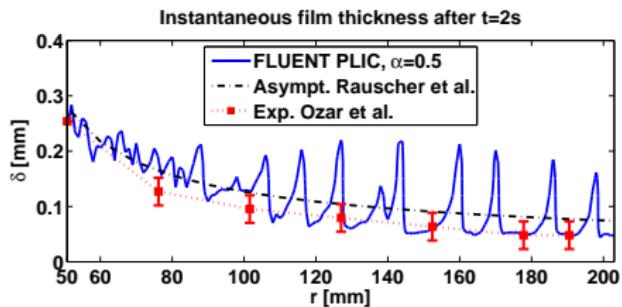


Thomas et al., 1991 [4],
Ozar et al., 2003 [5]

Test case I - Instantaneous film thickness $(\omega = 200\text{rpm}, Q = 7\text{lpm})$



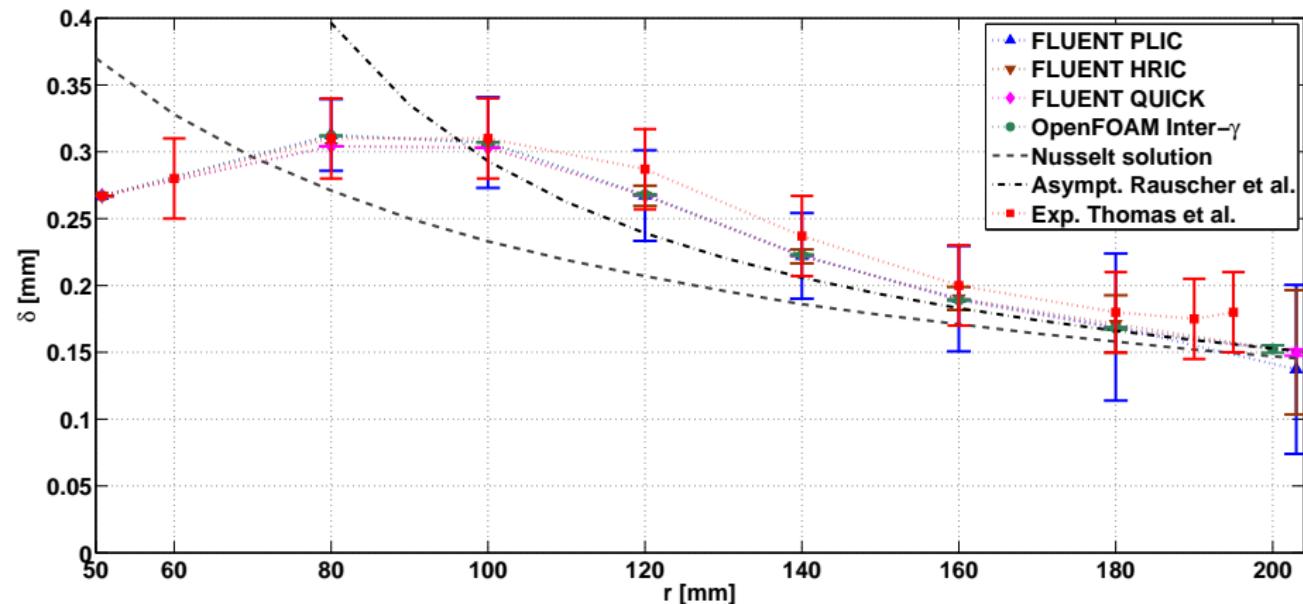
Test case II - Instantaneous film thickness $(\omega = 300\text{rpm}, Q = 3\text{lpm})$



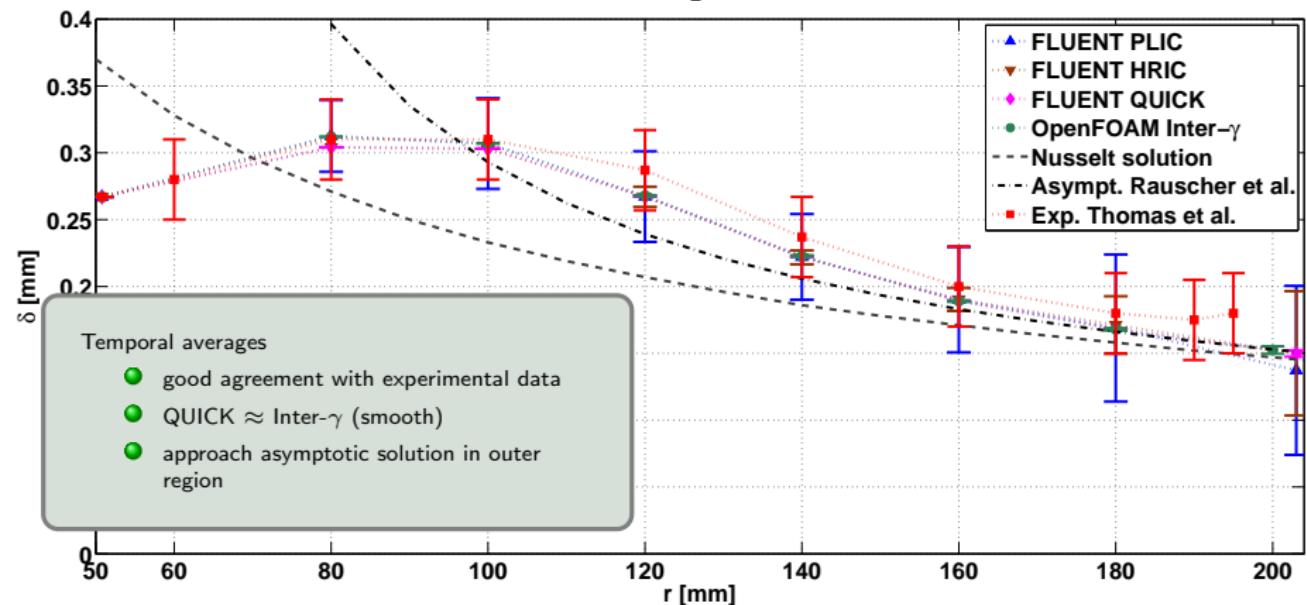
Test case I - Time averaged values

 $(\omega = 200\text{rpm}, Q = 7\text{lpm})$

Test case I
 $\omega=200\text{rpm}, Q=7\text{lpm}, v_L=1\times 10^{-6}\text{m}^2/\text{s}, \theta=10\text{deg}$



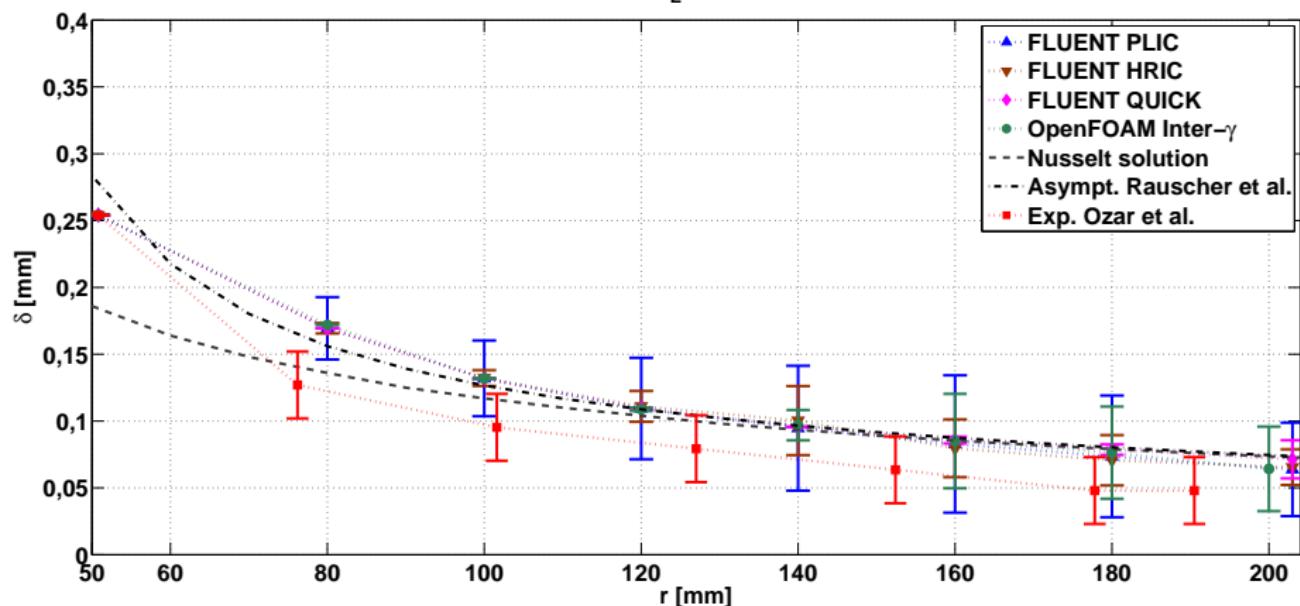
Test case I - Time averaged values

 $(\omega = 200\text{rpm}, Q = 7\text{lpm})$ Test case I
 $\omega=200\text{rpm}, Q=7\text{lpm}, v_L=1\times 10^{-6}\text{m}^2/\text{s}, \theta=10\text{deg}$ 

Test case II - Time averaged values

 $(\omega = 300\text{rpm}, Q = 3\text{lpm})$

Test case II
 $\omega=300\text{rpm}, Q=3\text{lpm}, v_L=0.66 \times 10^{-6} \text{m}^2/\text{s}, \theta=10\text{deg}$

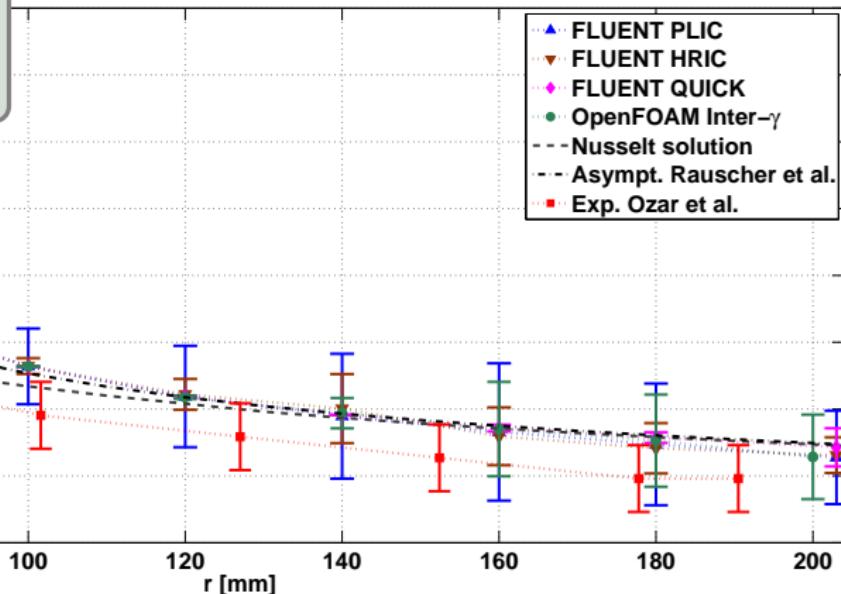


Test case II - Time averaged values

 $(\omega = 300\text{rpm}, Q = 3\text{lpm})$

- Exp. data overpredicted
- HRIC, Inter- γ : enhanced waviness
- Smaller Ro^2 ($\omega \uparrow, Q \downarrow$)

Test case II
 $300\text{rpm}, Q=3\text{lpm}, v_L=0.66 \times 10^{-6} \text{m}^2/\text{s}, \theta=10\text{deg}$



Conclusions 1/2

Comparison: OpenFOAM - FLUENT

- Both CFD codes produce comparable *time averaged* values
- Significant differences in *instantaneous* values associated with surface tracking method
 - PLIC: interface highly distorted
 - HRIC, Inter- γ and QUICK show smoother solutions with smaller waviness
- Sensitivity of instantaneous results to surface tracking schemes requires further investigations



Conclusions 2/2

Comparison against Experiments & Asymptotic solution

- Asymptotic solution:
good agreement of time averaged values in both cases
- Experimental:
 - Good agreement for Test case I
 $(\omega = 200\text{rpm}, Q = 7\text{lpm})$
 - Overpredictions for Test cases II
 $(\omega = 300\text{rpm}, Q = 3\text{lpm})$
 - possibly enhanced 3d-effects?
 - influence of measurement technique?



-  J. Rauscher, R. Kelly, J. Cole, An asymptotic solution for the laminar flow of thin films on a rotating disk, *Appl. Mechanics* 40 (1973) 43–47.
-  R. Scardovelli, S. Zaleski, Direct numerical simulation of free-surface and interfacial flow, *Annu. Rev. Fluid Mech.* 31 (1999) 567–603.
-  C. Hirt, B. Nichols, Volume of fluid VOF method for the dynamics of free boundaries, *Journal of Computational Physics* 39 (1981) 201–225.
-  S. Thomas, A. Faghri, W. Hankey, Experimental analysis and flow visualization of a thin liquid film on a stationary and rotating disk, *Journal of Fluids Engineering* 113 (1991) 73–80.
-  B. Ozar, B. Cetegen, A. Faghri, Experiments on the flow of a thin liquid film over a horizontal stationary and rotating disk surface, *Experiments in Fluids* 34 (2003) 556–565.